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# ASSESSMENT OF STABILITY OF NATURAL CIRCULATION IN THE FIRST CIRCUIT OF SMALL MODULAR REACTORS

В. Ковальчук, І. Козлов, О. Чулкін, К. Карчев, В. Мілєв. Оцінка стійкості природної циркуляції у першому контурі малих модульних реакторів. Робота містить результати дослідження питання впливу архітектури та робочих параметрів водоводяної модульної реакторної установки на її функціонування в режимі природної циркуляції теплоносія першого контуру. Метою дослідження є оцінка можливості організації режиму природної циркуляції в першому контурі модульного реактора із заданими структурою, геометрією та тепловими показниками. У світі на різних етапах розробки перебуває понад 50 проектів реакторів малої потужності, призначених для задоволення енергетичних потреб регіонів різного рівня ресурсозабезпеченості. До передбачуваних переваг модульних конструкцій, поряд з можливістю їхнього серійного виробництва, відносять гнучкість режимів експлуатації та підвищену безпеку, обумовлену застосуванням природної циркуляції теплоносія в першому контурі. Методологія базується на класичних принципах балансу сил, що діють на систему, в стаціонарному стані, якою є замкнутий контур реакторної установки в процесі експлуатації. Необхідні характеристики компонентів контуру (реактора та парогенератора) отримані в результаті їх розрахунків стосовно вибраного базового модуля SMR 160. Технологічна структура енергетичного об'єкта отримана масштабним аналізом ілюстративних матеріалів. Порівняння розрахункових значень рушійного напору (19412 Па) та сумарних втрат у контурі (27302 Па) показало, що для організації стійкого режиму природної циркуляції необхідно шляхом спільної варіації параметрів архітектури та режимних характеристик контуру оптимізувати структуру установки, що забезпечує гнучкі режими експлуатації. Результати дослідження підтвердили доцільність застосування принципу балансу діючих сил з метою оцінки можливості організації природної циркуляції у замкнутому контурі. У роботі запропоновано методику передпроєктного спільного аналізу архітектури енергетичного модуля та його робочих параметрів для забезпечення надійної та безпечної експлуатації ядерної установки в автономних умовах.

Ключові слова: модульні реактори, природна циркуляція, режим експлуатації, рушійний напір, гідравлічний опір, контур циркуляції

V. Kovalchuk, I. Kozlov, O. Chulkin, K. Karchev, V. Miliev. Assessment of stability of natural circulation in the first circuit of small modular reactors. The work contains the results of the study of the influence of the architecture and operating parameters of the water-based modular reactor plant on its functioning in the mode of natural circulation of the coolant of the first circuit. The purpose of the study is to assess the possibility of organizing a natural circulation regime in the first circuit of a modular reactor with the given structure, geometry and thermal parameters. In the world, there are more than 50 low-power reactor projects at various stages of development, designed to meet the energy needs of regions with different levels of resource availability. The expected advantages of modular designs, along with the possibility of their serial production, include the flexibility of operating modes and increased safety due to the use of natural circulation of the coolant in the first circuit. The methodology is based on the classical principles of the balance of forces acting on the system in a steady state, which is the closed loop of the reactor plant during operation. The necessary characteristics of the circuit components (reactor and steam generator) were obtained as a result of their calculations for the selected basic module SMR 160. The technological structure of the energy facility was obtained by a large-scale analysis of illustrative materials. Comparison of the calculated values of the driving pressure (19412 Pa) and total losses in the circuit (27302 Pa) showed that to organize a stable natural circulation mode, it is necessary to optimize the structure of the installation by jointly varying the parameters of the architecture and the operating characteristics of the circuit, which ensures flexible operating modes. The results of the study confirmed the feasibility of applying the principle of balance of forces to assess the possibility of organizing natural circulation in a closed loop. The paper proposes a methodology for pre-design joint analysis of the architecture of the power module and its operating parameters to ensure reliable and safe operation of a nuclear facility in autonomous conditions.

Keywords: modular reactors, natural circulation, operating mode, driving pressure, hydraulic resistance, circulation circuit

## Introduction

The last two decades have been characterized by close attention to the development of new technologies that contribute to overcoming the problems of optimal supply of regions with vital resources. An important position in this direction is nuclear energy. Developers of new reactor technologies claim that the future of nuclear power lies in small modular reactors (SMR), which have a number of advantages over large-capacity power units. More than 50 projects of low-power reactors are in various stages of development in the world [1].

Interest in them is due to a number of expected advantages: modular design, serial production of power units, operational flexibility and increased safety. This is due to the use of natural circulation of the coolant in the first circuit.

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## Analysis of the state of the problem

In water-water type reactors, the circulation of the coolant is usually carried out forcibly. Of great practical interest is the possibility of reactor operation with natural coolant circulation (NCC), which is especially important for emergency (in case of cessation of coolant supply to the active zone by the pump) and regular cooling modes, as well as power increase [2, 3].

In some SMR projects, the principle of natural circulation is expected to be applied. For example, in the CAREM-25 (Argentina) and NuScale (USA) projects, in normal operation mode, full natural circulation of the coolant of the first circuit is used in the absence of main circulation pumps [4].

Natural circulation is understood as the ability of the liquid to circulate continuously in the system under the action of a single driving force – "thermal pressure", which occurs when the density of the heat carrier changes under the influence of the introduction and removal of thermal energy in the circuit. A liquid system intended for natural circulation must have a heat source and a heat sink, which can be used as an active zone and a steam generator, respectively.

In reactors operating on the principle of natural circulation, the combination of neutron-physical and thermo-hydraulic phenomena is important. As part of the joint efforts of the IAEA to promote international cooperation aimed at improving the safety of nuclear power plants, a program was implemented to develop methods for evaluating the operation of the passive safety system in advanced reactors, which notes the advantages and disadvantages of natural circulation systems (Table 1), [5].

Table 1

Advantages and disadvantages of natural circulation systems

Advantages	Disadvantages		
The lowest cost due to simplicity	Low driving force		
Lack of pumps	Lower maximum power in each channel		
The possibility of improving the current distribution in the active zone	Potential instability		
The best two-phase characteristics depending on the power	Low critical heat flux		
Great thermal inertia	The need for complex start-up procedures		

The possibility of providing several passive systems based on the principle of natural circulation, in addition to active systems, is considered in the operated nuclear reactors. This will not only increase the operational safety of reactors, but also prevent hypothetical severe accidents.

One of the possible emergency situations that can occur during the operation of NPP power units is the main circulation pumps (MCP). In such situations, emergency shutdown of the reactor and closure of the turbine stop-control valves are required. The reactor itself is then transferred to the natural coolant circulation mode (NCC) in the primary circuit, which will ensure the release of excess energy during the process of refrigeration. With normal parameters of the coolant at the entrance to the reactor after shutdown ( $t_{in}$ =288 °C), it is possible to ensure the removal of thermal power from the reactor plant, which does not exceed 7...10% of the nominal one, which is quite enough to ensure reliable cooling of the reactor [3]. It is expedient to increase the heat output from the reactor in the mode of natural circulation of the coolant in order to ensure the operation of the power unit when the main circulation pump is de-energized in the energy mode (albeit at a reduced power level) with the delivery of electricity to the network [3].

The April 2023 agreement between Energoatom NAEC and Holtec International (USA) envisages the construction of up to 20 nuclear power units with SMR-160 reactors in Ukraine, with the implementation of the first pilot project and reaching the minimum regulated power of the reactor and connecting to the grid by March 2029 year. Therefore, the above-mentioned module was chosen as the object of research.

## The purpose and objectives of the research

The purpose of the presented material is to assess the possibility of organizing a natural circulation regime in the first circuit of a modular reactor with the given structure and thermal parameters.

To achieve the goal, it is necessary to solve the following tasks:

- Obtaining the geometric structure of the first contour of the selected module;

- Obtaining the calculated structural and technological indicators of the elements of the first circuit of the selected module;
  - Calculation of thermohydraulic parameters of the coolant in the circuit.

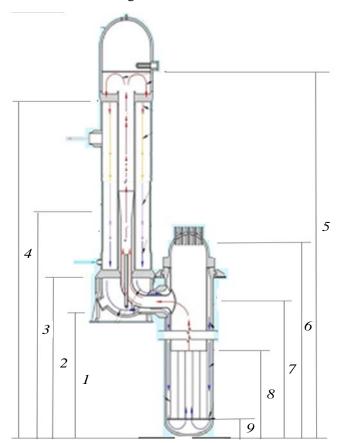
## Object and method of research

The Holtec SMR-160 modular reactor is a non-boiling, pressurized light water (PWR) system. Electric power 160 MW, thermal power – 525 MW [6].

The first circuit is supposed to be implemented in the form of a monoblock, which includes a rigidly connected reactor and a steam generator. A lead is made from the space above the active zone, which runs along the axis of the steam generator to its upper inlet chamber, where the flow is expanded and distributed through the heat exchange pipes. Further, the flow is collected in the lower chamber of the steam generator and enters the space around the active zone through the annular pipeline (Fig. 1). The natural circulation of the coolant is carried out under the influence of thermal pressure between the rising and lowering channels of the circuit.

The characteristics of the reactor and steam generator necessary for the analysis were obtained in previously performed thermohydraulic calculations (Tables 2, 3, 4) [7, 8, 9].

A large-scale analysis of the illustrations accompanying the available descriptions of the module created a table of heights of individual elements of the module (Table 2).



**Fig. 1.** Diagram 1 of the circuit of the module: *I* – The lower limit of the output chamber of the steam generator; *2* – Lower pipe board; *3* – Border of economizer and evaporation areas; *4* – Upper pipe board; *5* – Pressure compensator membrane; *6* – The upper level of the space above the active zone; *7* – The level of removal from the space above the active zone to the steam generator; *8* – Upper level of the active zone; *9* – The lower level of the active zone

Table 2
Limit heights of monoblock elements

Emit neights of monoblock elements					
No	Levels	Height, m			
1	The lower limit of the	9.9			
	collecting chamber				
2	The lower grill of the	13.3			
	steam generator				
3	Border economizer-	13.3			
	evaporator				
4	The upper grid of the	33			
	steam generator				
5	Compensator mem-	35			
	branes				
6	The upper limit of the	13.8			
	supra reactor space				
7	Discharge to the riser	9.9			
	area, which is not				
	heated				
8	The upper limit of the	2.7			
	active zone				
9	The lower limit of the	0.6			
	active zone				

In order to assess the possibility of implementing circulation, as well as the safety and reliability of the module operation, a comparison of the calculated thermohydraulic modes and the spatial structure of the circuit was made.

The condition for the stability of the stationary mode of natural circulation in a closed circuit is the equality of the driving pressure  $\Delta p_{\rm mp}$  and the hydraulic resistance  $\Delta p_{\rm hr}$  of the circuit [10, 11].

The moving pressure  $\Delta p_{\rm mp}$  is directly

proportional to the difference in the densities of the heat carrier in the rising ("hot"  $\rho_h$  ) and lowering ("cold"  $\rho_c$ ) sections of the circulation circuit and the difference  $\Delta H$  of the height marks of the sections with heat supply (active zone) and removal (steam generator):

Table 3

$$\Delta p_{\rm mp} = g(\rho_c - \rho_h) \Delta H \,, \tag{1}$$

the hydraulic resistance  $\Delta p_{\rm mp}$  is proportional to the square of the flow  $G_i$  of the coolant through the circuit:

$$\Delta p_{\rm hr} = \sum_{i=1}^{n} \xi_i \frac{G_i^2}{2\rho_i F_i^2} \,, \tag{2}$$

where  $\xi_i$  – coefficient of resistance;

 $\rho_i$  – is the density of the heat carrier;

 $F_i$  is the area of the circuit section.

Taking into account (1, 2), the condition for the stability of the natural circulation regime takes the form:

$$g(\rho_{c} - \rho_{h})\Delta H = \sum_{i=1}^{n} \xi \frac{G_{i}^{2}}{2\rho_{i}F_{i}^{2}},$$
 (3)

where g – acceleration of free fall, m/s<sup>2</sup>;

 $\rho_{x}$ ,  $\rho_{g}$  – heat carrier density in the rising and lowering sections of the main circulation circuit (MCC), kg/m<sup>3</sup>;

 $\Delta H$  – the difference in height between the centers of the heat exchange surface of the steam generator and the active zone of the reactor, m;

 $\xi_i$  – coefficient of hydraulic resistance of the ith section of the MCC;

 $G_i$ ,  $\rho_i$  – flow rate (kg/s) and heat carrier density (kg/m<sup>3</sup>) at the i<sub>th</sub> section of the MCC;

 $F_i$  – the area of the cross-section of the *i*-th section of the MCC, m<sup>2</sup>.

Calculation characteristics of the active zone

Units of Characteristics Designation Value under load measurement.  $N_{\rm p}^{\rm AV}$ Average specific volumetric load  $MW/m^3$ 65 100 525 525 Heat capacity  $N_{\rm H}$ MW 525 The density of the coolant under operating 758.7 758.7 758.7 kg/m<sup>3</sup>  $\rho_{\rm az}$ parameters Volume of the active zone  $V_{az}$  $m^3$ 8.6 6.6 5.6 The diameter of the active zone 2.33 2.13 2.02 m The height of the active zone  $H_{az}$ 2.03 1.85 1.76 m Speed required for heat removal 0.64 0.77 0.86  $w_{\rm az}$ m/s Number of fuel assemblies piece 53 58 61 Leveling component of hydraulic resistance  $\Delta p_1$ kPa 13.8 12.6 11.9 Hydraulic friction losses kPa 0.37 0.47 0.54  $\Delta p_{\mathrm{f}}$ Hydraulic losses from local resistances 0.60 0.86 0.92 kPa  $\Delta p_{\rm lr}$ Coolant pressure losses in a medium-loaded cell kP 14.7 13.9 13.4  $\Delta p_{\Sigma}$ 

To estimate the maximum driving pressure in the circuit with natural circulation according to the ratio (1), the difference in height marks between the centers of the heat exchange surface of the steam generator and the active zone of the reactor is necessary.

Active forces at the entrance to the active zone:

- from the side of the down channel,

$$F_0 = g \rho_{sg}^c \Delta H_{sg} + g H_2 \cdot \rho_{sg}^{esg} - \Delta p_0; \tag{4}$$

- from the side of the lifting channel,

$$F_L = g(\rho_{so}^c \Delta H_{so} + (H_5 - H_8)\rho_{so}^{esg}) + \Delta p_L, \tag{5}$$

where  $\rho_{sg}^{c}$  – the density of the heat carrier in the center of the heat exchange surface of the steam generator:

 $\Delta H_{\rm sg}$  is the height of the heat exchange surface of the steam generator;

 $\rho_{\scriptscriptstyle sg}^{\scriptscriptstyle esg}$  – the density of the heat carrier at the exit from the steam generator;

 $\Delta H_2$  – the height of the lower boundary of the heat exchange surface of the steam generator;

 $\Delta p_0$  – the sum of the hydraulic losses in the lowering channel;

 $(H_5-H_8)$  is the height of the section of the lifting channel that is not heated;

 $\Delta p_l$  is the sum of hydraulic losses in the lifting channel.

Table 4
Calculation characteristics of the steam generator

Characteristics	Designation	Units of measurement	Values at the speed of the coolant		
heat carrier flow through the first circuit through the steam generator	$G_{HC}$	kg/s	1479	1479	1479
heat capacity of the evaporator	$N_{HE}$	MW	443.4	443.4	443.4
thermal capacity of the economizer	$N_{TE}$		81.63	81.63	81.63
coolant speed	$w_{ m HC}$	MW	1	1,6	2.6
the number of tubes on the surface	$n_{ m NT}$	m/s	22540	14080	8667
Number of steam generator modules	$n_{M{ m SG}}$	piece	8/2817	8/1761	8/1083
heat exchange surface of the evaporator	$S_{EV}$	piece	1439	1119	884
the length of the tube of the evaporation section	$L_{B\Pi}$	$m^2$	13.01	16.19	20.78
average speed	$w_{B\Pi}$	m	0.89	1.43	2.32
heat exchange surface of the economizer	$S_{EK}$	m/s	127	101	82
the length of the tube of the economizer section	$L_{EK}$	$m^2$	1.03	1.3	1.72
total calculated heat exchange area	$S_{ m HE}$	m	1566	1219	965
with a margin	$S_{ m HEm}$	$m^2$	1801	1402	1110
the total length of the tube, taking into account the margin factor	$L_{ m HEm}$	m	16.34	20.36	26.19
hydraulic losses due to friction	$\Delta p_{ m f}$	m	16467	52513	178407
turn 90 degrees	$arvarepsilon_l$	Pa	0.2	0.2	0.2
entrance to the tubes from the distribution chamber	$arvarrow_l$	_	0.5	0.5	0.5
exit to the assembly chamber	$arvarepsilon_l$	-	1.0	1.0	1.0
entrance to the outlet pipe from the collector	$arvarepsilon_l$	_	0.5	0.5	0.5
hydraulic losses on local supports	$\Delta p_{ m ls}$	_	825	2112	5577
hydraulic resistance of the first circuit	$\Delta p_{ m hr}$	Pa	17292	54625	183984

Equality of forces in any section of the circuit ensures a stationary mode of natural circulation:

$$g(\rho_{sg}^{c}\Delta H_{sg} + H_{2}\rho_{sg}^{esg}) - \Delta p_{l} =$$

$$= g(\rho_{az}^{c}\Delta H_{az} + (H_{5} - H_{8})\rho_{az}^{ef}) + \Delta p_{0} + \Delta p_{Fr},$$
(6)

where  $\rho_{az}^c$  - the density of the heat carrier in the center of the heat exchange surface of the active zone;

 $\Delta H$  – the height of the heat exchange surface of the active zone;

 $\rho_{az}^{ef}$  – the density of the heat carrier at the exit from the active zone;

Head loss in the circuit, caused by the switching features of the steam generator and the active zone, which takes into account the indicator of hydraulic losses due to friction  $\Delta p_{\rm Fr}$ .

After transforming (6) into form (7):

$$g(\rho_{so}^{c}H_{so} + H_{2}\rho_{so}^{esg} - \rho_{az}^{c}H_{az} - (H_{5} - H_{8})\rho_{az}^{ef}) = \Delta p_{l} + \Delta p_{0} + \Delta p_{Fr},$$
(7),

we will get an expression, the left part of which reflects the effective heat pressure, and the right part – the sum of hydraulic losses in the circuit.

#### The results of the research

Substitution of data from Tables 2, 3 and 4 allows you to obtain the result of checking the condition of the existence of a stationary mode of natural circulation:

$$9.81(758.73(33-13.3)+13.3\cdot825.42-758.73(2.7-0.6)-(35-2.7)692.04)=13923.15+9949.38;$$
  
thrust  $\Rightarrow 19.412 \,\mathrm{kPa} = total \,\mathrm{losses} \Rightarrow 23.872 \,\mathrm{kPa}.$ 

The calculated driving pressure was 19412 Pa, and the total losses in the circuit reach 27302 Pa. With the obtained ratio of driving pressure and losses, the occurrence of natural circulation is excluded. It follows from (7) that with unchanged thermo-hydraulic parameters, the ratio of head and losses depends on the difference in the height of the average temperatures in the rising and lowering channels, which are constructively determined by the values of  $H_2$  and  $H_5$ . Variations of their values showed that an increase in the values accepted in the calculation by 2.7...2.8 m allows to achieve the necessary balance of driving pressure (27302 Pa) and losses (27302 Pa), and, therefore, to realize the mode of natural circulation. The size of the variation does not exceed 7% of the total height of the block and can be attributed to the inaccuracies of the illustrative material used in the large-scale analysis.

#### **Conclusions**

- The geometric structure of the first contour of the Holtec SMR-160 module was obtained by large-scale analysis of existing illustrations;
- The structural and technological indicators of the active zone, steam generator and switching elements of the first circuit of the Holtec SMR-160 module were obtained;
- The thermal-hydraulic parameters of the coolant in the circuit were calculated and analyzed. It has been proven that the chosen architecture of the circuit allows to organize a stable mode of natural circulation in the circuit of the SMR 160 modular reactor.

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